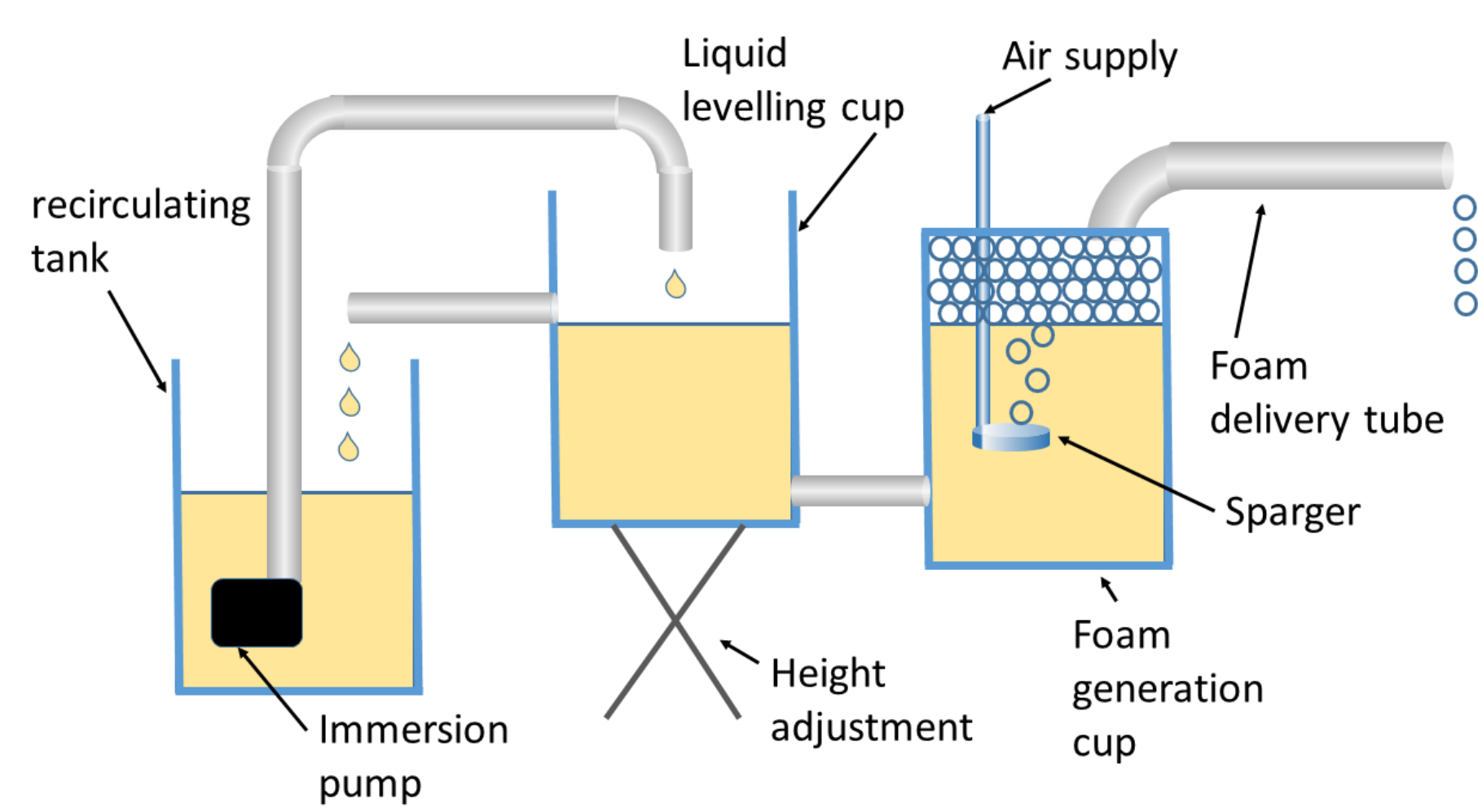




## Bench scale fire extinguishing test

Foams are a very effective and widely used method to extinguishing fires. They are especially useful to extinguish large pool fires of liquid fuels. To test the capabilities of newly developed foams and foam components, a bench scale fire extinguishing test has been developed at NRL. The sophisticated design of the foam generator guarantees highly similar foam properties at different foam flow rates [3]. Extinction time data of > 200 extinguishing tests with > 80 different foam solutions at many different foam flow rates has been generated at NRL with the bench scale test. The goal is to find a simple mathematical model which correlates material data of the foam solution to experimental results to further cut back the number of fire tests necessary for the development of a new foam.

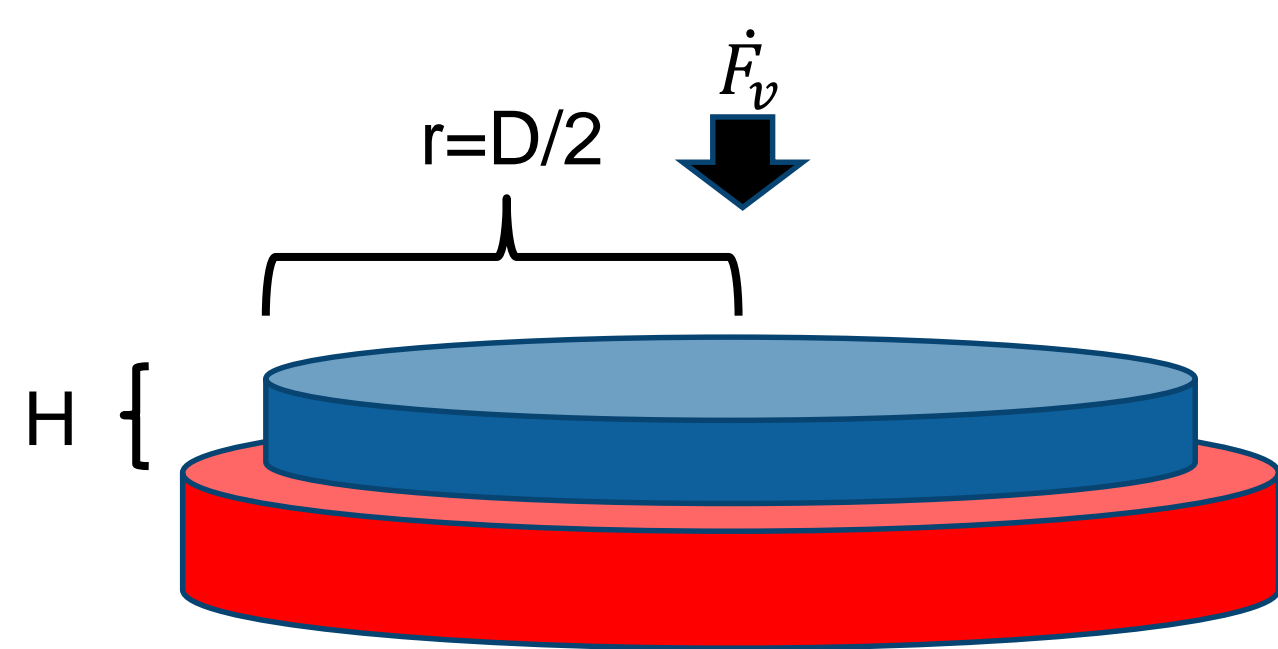


Foam generator for bench scale test



RefAFFF bench scale fire test on heptane pool  
502W Formulation fire test on heptane pool

## Mathematical model for the bench scale fire extinguishing test



### Model assumptions

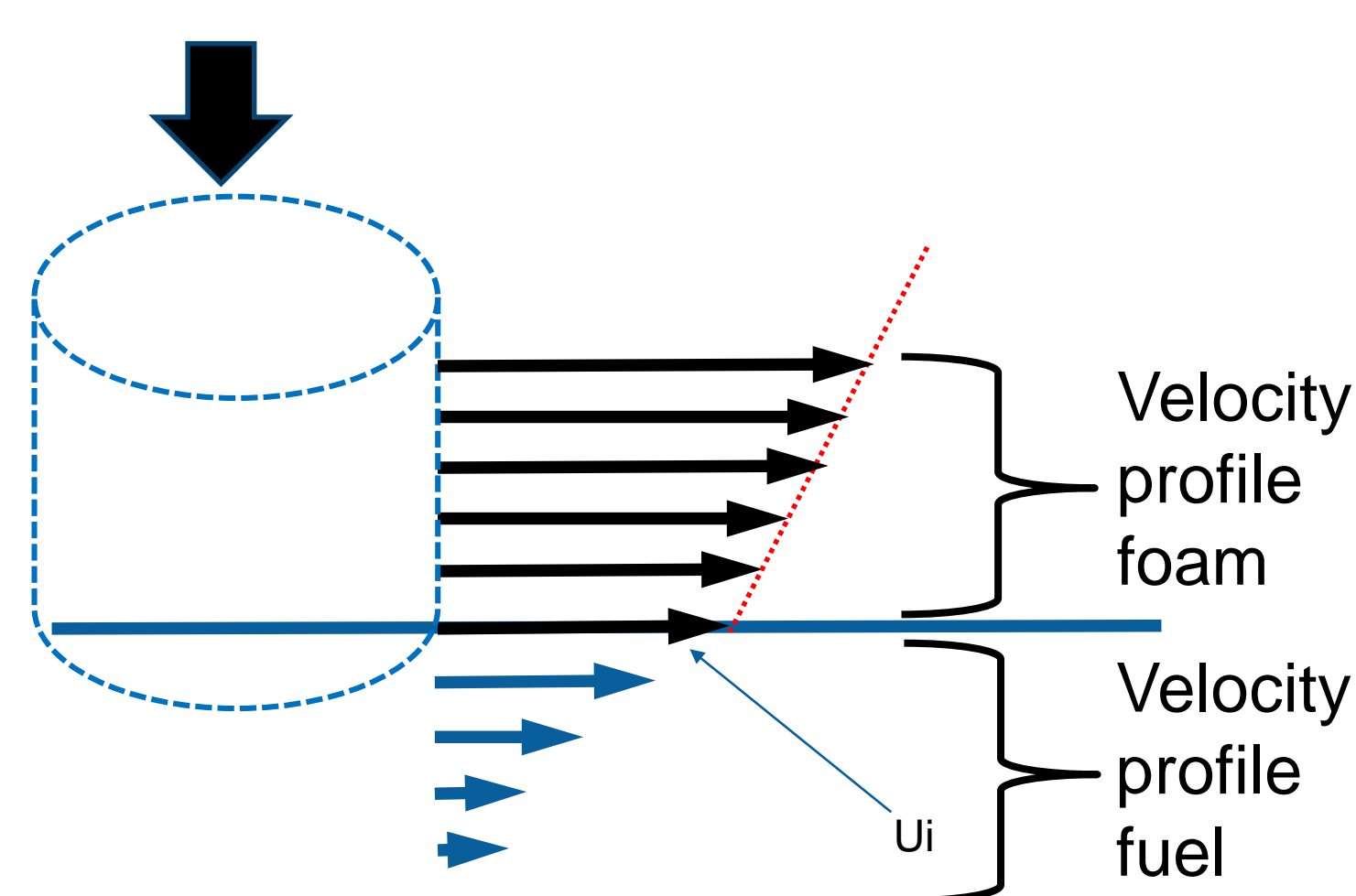
- Constant height  $H_f$  of cylindrical foam layer with variable diameter  $D$
- Uniform density of foam
- No drainage
- Constant mass loss  $R_{dfu}$  due to fuel/foam interaction proportional to area covered by foam
- Mass loss due to fire/foam interaction at constant rate  $R_{dfi}$  proportional to circumference of foam layer
- Foam is applied in the center of the circular pan with constant rate  $F_v$
- Fire is extinguished when diameter of foam is equal or larger than diameter of the cylindrical pool

$$\rho\pi H_f D \frac{dD}{dt} = \rho F_v - \rho \frac{\pi}{4} D^2 R_{dfu} - \rho\pi H_f D R_{dfi}$$

## Couette Flow Assumption

One additional assumption is that the velocity of the foam at the fuel/foam interface  $U_i$  is a constant at a given radius  $R_t$  around the foam input location. It changes with height at a constant rate. By integrating over the velocity profile and solving for  $H_f$  the height of the foam on top of the pool can be determined:

$$F_v = \int_0^{H_f} 2\pi r u(h) dh \Rightarrow H_f = -\frac{\mu u_i}{\tau} + \sqrt{\frac{\mu^2 u_i^2}{\tau^2} + \frac{\mu F_v}{2\pi R_t \tau}}$$

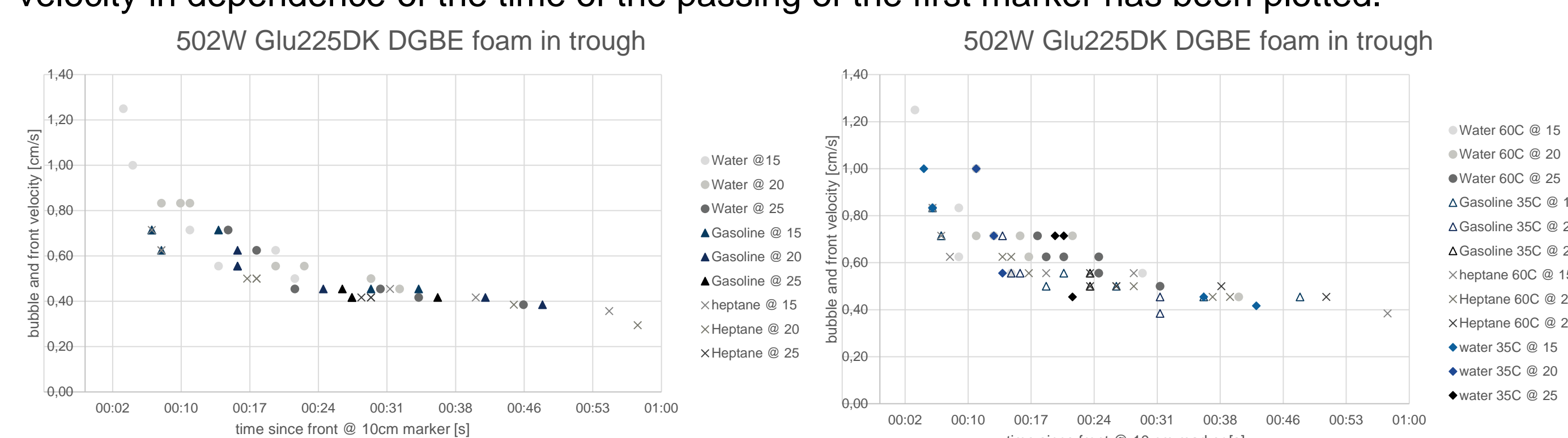


## Validation in 1D Trough

In a one dimensional trough, if the top layer is the fastest layer, the tip of the foam should have the same velocity as all other points at the top of the foam. In the other case, the tip of the foam should move faster than other points on the top of the foam, with the difference increasing with the distance from the tip of the foam. The same holds for the time domain.



To measure the speed of different points at the top layer of the foam, the movement of bubbles at the top of the foam has been observed. The times when the bubbles passed the 10 cm mark, 15 cm mark etc up to the 35 cm mark have been recorded, when applicable. To compute the velocity of the foam bubbles, the travel time between two marks has been divided by the distance (5 cm). The resulting velocity in dependence of the time of the passing of the first marker has been plotted.



## Fitting model to experimental data

Model predicts extinguishing times  $t_{ex}$  at given foam flow rates  $F_v$  using a set of parameters ( $R_{dfi}$ ,  $R_{dfu}$ ) named P:

$$t_{model ij} = Model(F_{v ij}, P_k j)$$

For the foam solution  $j$ , different foam flow rates  $i$  and the set of parameters P

To fit the model to the experimental data, the sum of squares has to be minimized. This is done by adjusting the parameters P:

$$\sum_i (t_{ex ij} - t_{model ij})^2$$

$$\sum_i (t_{ex ij} - Model(F_{v ij}, P_k j))^2$$

For one foam solution  $j$

Initial guesses for the set of parameters P could be found by investigating limit cases:

$$R_{dfu} \text{ initial guess} = \frac{4F_v}{\pi D_{pool}^2}$$

With the highest foam flow rate  $F_v$  without successful extinguishment

$$u_i \text{ plug initial guess} = \frac{(D_{pool}^2 - 0.01^2 m^2)}{16R_t t_{ex plug}}$$

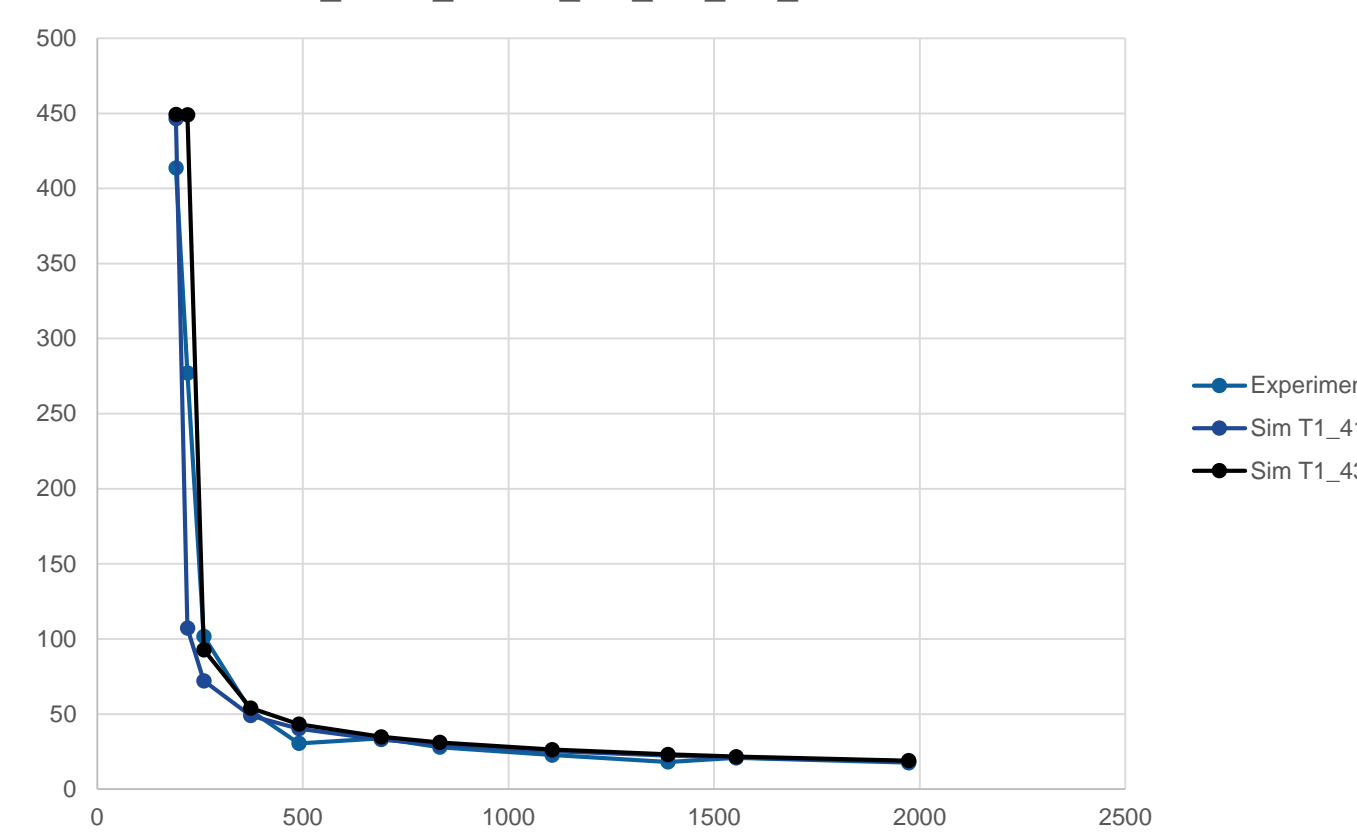
With the extinguishing time  $t_{ex}$  at the highest foam flow rate

$$R_{dfi} \text{ initial guess} = \frac{F_v - \frac{\pi}{4} D_{pool}^2 R_{dfu}}{\pi H_f D_{pool}}$$

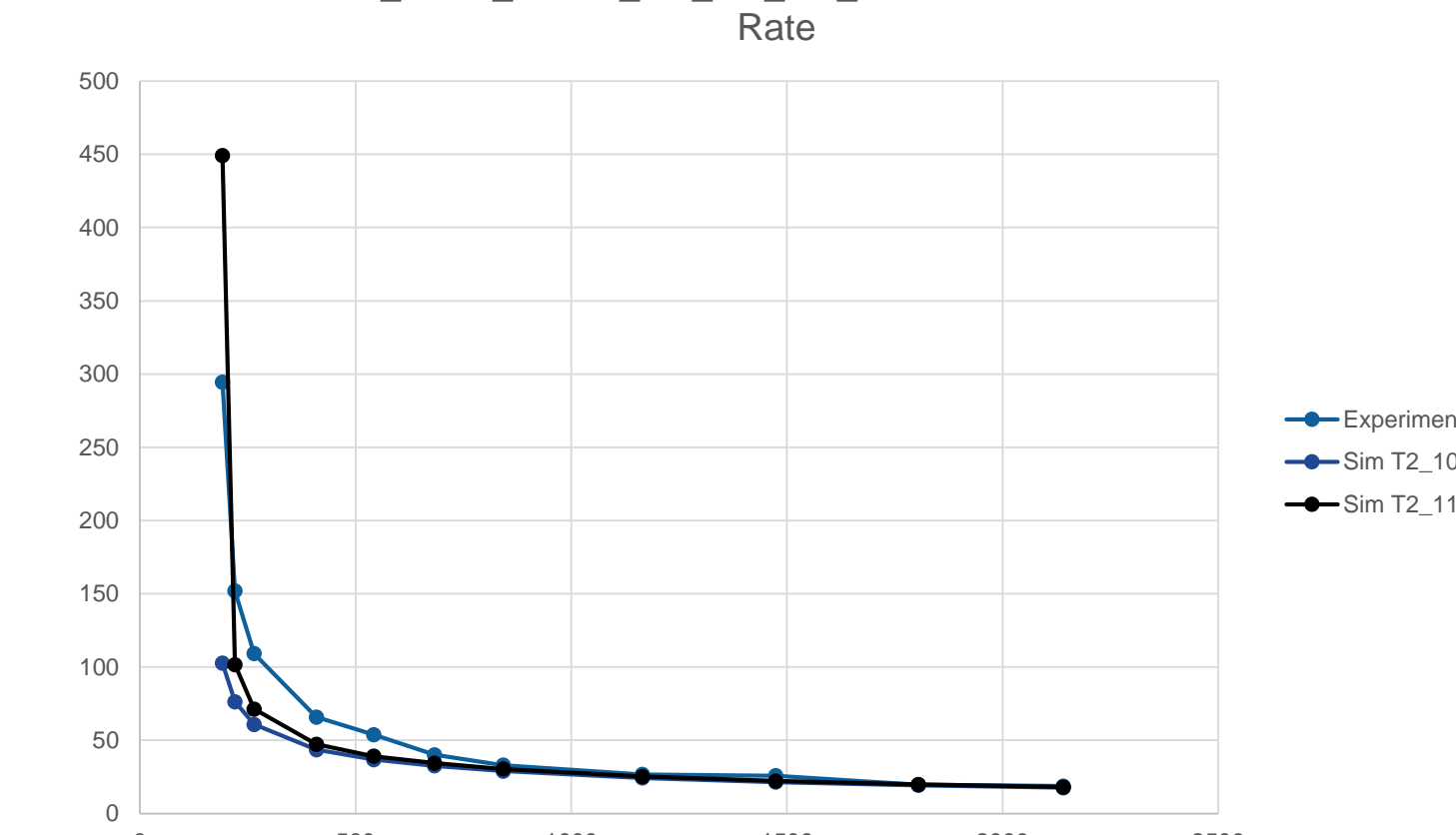
With the median foam flow rate  $F_v$

Depending on the set of data chosen, the quality of the fit of the model to the experimental data varies.

D502W\_G225\_DGBE\_0.2\_0.3\_0.5\_T1 Ext Time vs Flow Rate

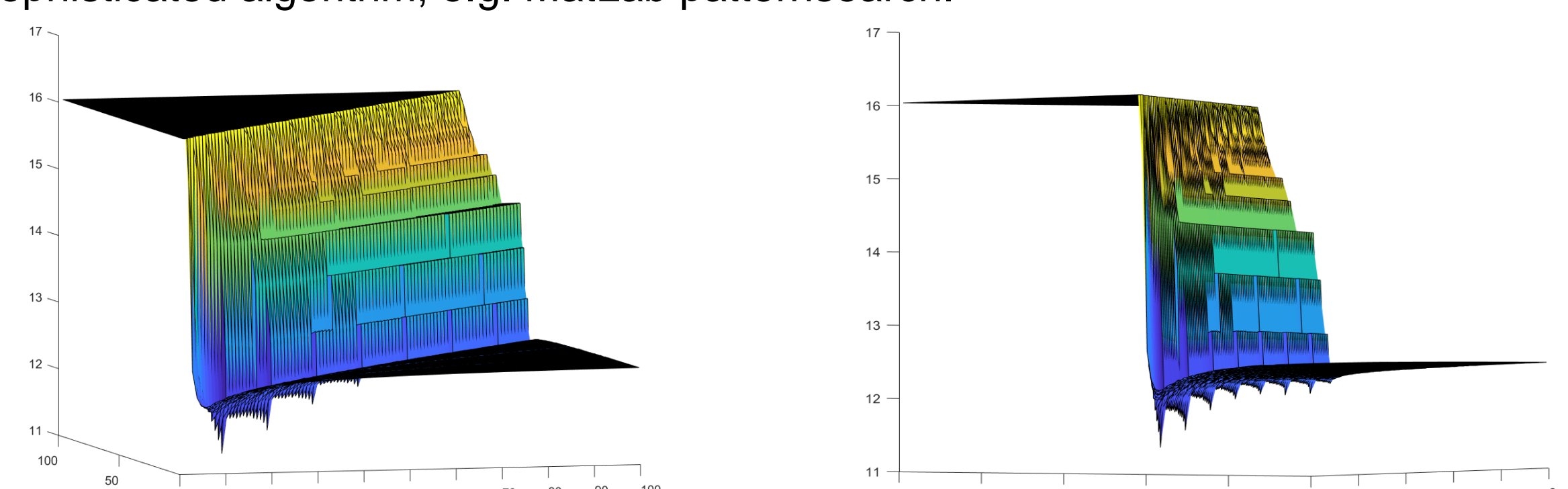


D502W\_G225\_DGBE\_0.2\_0.3\_0.5\_T2 Ext Time vs Foam Flow Rate



## Numerical Challenges:

The residual for a set of parameters with  $R_{dfu} = 2.333 \cdot 10^{-5}$  m/s,  $R_{dfi} = 0 \dots 5 \cdot 1.8 \cdot 10^{-3}$  m/s,  $U_i = 0 \dots 5 \cdot 0.02$  m/s for the model to 0.2% D502W 0.3% G225 0.5% DGBE\_T1 foam on heptane has been plotted, with  $\ln(\text{residual})$  as the Z-axis and values proportional to  $R_{dfi}$  and  $U_i$  on the X and Y axis. The resulting figures below show that the minimum of the residual is extremely difficult to find, because it is 'hidden' in a narrow valley with many local minima which confuse the minimum seeking algorithm. High numerical effort is necessary to identify the minimum provided by a sophisticated algorithm, e.g. MatLab patternsearch.



## Results

Correlation  $R_{dfu}$  to Degradation time

	5-26	6-7a	6-7b
All (63)	-0.36	-0.43	-0.41
CapStone (9)	-0.12	-0.26	0.05
502W (22)	-0.61	-0.55	-0.51

Correlation  $U_i$  to Cover time

	5-26	6-7a	6-7b
All (63)	-0.03	-0.03	-0.03
CapStone (9)	-0.78	-0.79	-0.69
502W (22)	-0.22	-0.13	-0.17

$$\frac{\sum_i (a_i - \bar{a})(b_i - \bar{b})}{\sqrt{\sum_i (a_i - \bar{a})^2 \sum_i (b_i - \bar{b})^2}}$$

Pearson correlation coefficient (PCC)

Presented are the results of three model runs. Because at 'no extinguishing' there is no associated extinguishing time, it is approximated by a very high extinguishing time. 999s in simulation 5-26 and 9999s in simulations 6-7a and 6-7b. To improve the stability of the model, in cases where for one mixture there was extinguishment in all fire tests, additional data was introduced with no extinguishment at flow rate close to zero (6-7a) and half of the lowest tested foam flow rate (6-7b). In all three cases the correlation between destruction by fuel predicted by model ( $R_{dfu}$ ) and foam degradation time is far better for the 502W subset. Similar, for the CapStone (an AFFF) subset the correlation between the velocity of the foam at the foam/fuel interface predicted by the model ( $U_i$ ) and experimental cover time is far better than the other correlations.

This correlation is remarkable, because only very simple assumptions about foam spread, foam destruction by the fuel and destruction by fire at the rim of the foam disc are made. Radiation to the top surface of the foam is ignored.

## References

1. Brian Y. Lattimer, Javier Trelles, Foam spread over a liquid pool, Fire Safety Journal 42 (2007) 249–264
2. Bror Persson and Martin Dahlberg, A Simple Model for Predicting Foam Spread Over Liquids, International Association of Fire Safety Science, Proceeding of the fourth international symposium, pp. 265-276
3. Ramagopal Ananth, Arthur W. Snow, Katherine M. Hinnant, Spencer L. Giles, John P. Farley, Synergisms between siloxane-polyoxyethylene and alkyl polyglycoside surfactants in foam stability and pool fire extinction, Colloids and Surfaces A 579 (2019) 123686